

# **OIL CIRCUIT OF A JACK FOR RISING OBJECT TO PRESET POSITION RAPIDLY**

## ***FIELD OF THE INVENTION***

5           The present invention relates to an oil circuit of a jack for rising  
an object to a preset position rapidly, which improve the device  
disclosed in U. S. Patent No. <sup>5,755,099</sup>~~5,577,099~~ so that the piston rod can attain  
a still state rapidly. The inner oil chamber of the piston rod is  
communicated to inner oil reservoir so that the inner and outer oil  
10       pressure of the oil tube are equal so that breakage is prevented.

## ***BACKGROUND OF THE INVENTION***

          In using a conventional hydraulic jack, despite of dummy load  
or loading condition, the handle must move repeated many times  
15       so as to drive the oil cylinder to work. However, since in dummy  
load or light load, it still needs to be driven repeated, much work  
is wasted. Therefore, the invention in U. S. Patent No. 5,755,099,  
"Hydraulic Circuit System for One Touch Jack and Its Structure",  
discloses a jack structure, in which the jack hydraulic loop circuit  
20       includes an oil inlet circuit, an oil return circuit and an overload  
protecting circuit.

          In the oil inlet circuit, the outer oil reservoir is communicated  
to the pumping oil chamber through a check valve. The pumping  
oil chamber is communicated to a oil guiding tube through another  
25       check valve, The oil guiding tube is embedded into the inner oil

chamber in the piston rod for forming as a loop. The pumping oil chamber is communicated to the inner oil reservoir of the piston rod through a sequential valve B. The outer oil reservoir is communicated to the inner oil reservoir of the hydraulic cylinder through a check valve. When a manual pump is in dummy load or light load, the oil circuit provides hydraulic oil to the inner oil chamber of the piston rod from the pump and through the oil guiding tube.

In the oil return circuit, the inner oil reservoir is communicated to the inner oil chamber of the piston rod through a check valve, then it further passes through a release valve to be communicated to the outer oil reservoir. When oil is unloaded and then it desires to the original position, the release valve <sup>can</sup> ~~and~~ be adjusted to a release position so that the oil return circuit is conducted.

In the overload circuit, an outer oil reservoir passes through a safety valve to be communicated to the oil chamber of a manual pump. When the pressure of the hydraulic cylinder is over a rated pressure, the safety valve will be conducted so that the overload protecting circuit will open automatically. In the aforesaid hydraulic loop system, the maximum effective oil storing amount of the pumping oil chamber of the manual pump is large than or equal to the maximum effective oil storing amount of the inner oil chamber in the piston rod so that the hydraulic jack may rapidly rise to a loading position in dummy or light load condition.

However, in above invention, for the check valves in the oil circuit and the oil channels, since when the piston rod move to a position in one touch, the oil guiding tube and the inner oil reservoir are in still condition. Thus, the inner and outer pressures of the oil guiding tube in the piston rod are unequal, the oil guiding tube must suffer from a large still load pressure. Therefore, the pressure-resistance of the oil guiding tube must be further improved, but this will increase the cost. If the pressure-resistance of the oil guiding tube is insufficient, it is possible to break out. Therefore, many considerations in safety are concerned for the prior art design.

### ***SUMMARY OF THE INVENTION***

Accordingly, the primary object of the present invention is to provide an oil circuit of a jack for rising an object to a preset position rapidly, wherein the oil inlet circuit of a hydraulic loop system is improved. An oil channel is installed between the inner oil chamber of the piston rod and the sequential valve. A check valve is installed between the oil channel and the sequential valve. An oil channel is installed between the sequential valve and the check valve for being connected to the inner oil chamber of the piston rod. By changing the positions of the check valve and the oil channel. When in the working conditions of dump load or light load, the sequential valve is closed, thus, the hydraulic oil may

enter into the inner oil chamber of the piston rod from the pumping oil chamber through the check valve so that the piston rod will rise rapidly to a still condition. In the still condition, since the check valve closes the oil channel, the sequential valve will open automatically so that the inner oil chamber of the piston rod is communicated to the inner oil reservoir. Thus the inner and outer oil pressures of the oil guiding tube in the piston rod are equal. Thus, no strong still load hydraulic pressure exists in the oil guiding tube within the inner oil chamber of the piston rod. BY this changing, the still load hydraulic pressure of the inner oil reservoir and the inner oil chamber of the piston rod can be adjusted equilibrium by the sequential valve. Therefore, the problem of breakage of the oil guiding tube of the piston rod and the high cost due to confinement in material are solved. Moreover, the sequential valve can be located outsides so that the sequential valve is adjustable outsides to a preset actuating pressure.

Another object of the present invention is to provide an oil circuit of a jack for rising an object to a preset position rapidly, wherein the sequential valve is located outsides with a 90 degrees of position shift, and a connecting oil channel is installed between a ball vale and the inner oil reservoir so that the adjusting nut of the sequential valve is locked to the outer wall of the rear seat of a jack. The sequential valve B can be located outsides so that the sequential valve is adjustable outsides to a preset actuating pressure.

The various objects and advantages of the present invention will be more readily understood from the following detailed description when read in conjunction with the appended drawing.

5                    **BRIEF DESCRIPTION OF THE DRAWINGS**

Fig. 1 illustrates a hydraulic circuit system according to the present invention.

Fig. 2 is a cross sectional view of the structure of a jack according to the present invention.

10           Fig. 3 illustrates displacement of the piston rod to its loading position in one step;

Fig. 4 illustrates further raising of the piston rod to support a load;

15           Fig. 5 illustrates displacement of raising arm and support plate by action of piston rod from a standstill position to a full raising position;

Fig. 6 is a sectional view of the sequence valve according to the present invention;

20           Fig. 7 is a schematic partial enlarged view of the embodiment shown in Fig. 6.

Fig. 8 is a schematic partial cross sectional view along A-A in Fig. 7.

***DETAILED DESCRIPTION OF THE PREFERRED  
EMBODIMENTS***

5        With reference to Figs. 1 and 2, the oil circuit of a jack for  
rising an object to a preset position rapidly of the present  
invention include an oil inlet circuit, an oil return circuit and an  
overload protecting circuit, and the inner oil reservoir 1, the outer  
oil reservoir 2, the pumping oil chamber 3 and piston rod 4 of a  
10    hydraulic cylinder 10.

      The oil inlet circuit is formed by the outer oil reservoir 2 of  
the hydraulic cylinder 10 being connected to the pumping oil  
chamber 3 through a check valve A1. The pumping oil chamber 3  
is connected to the inner oil reservoir 1 of the hydraulic cylinder  
15    10 through a sequential valve B. The outer oil reservoir 2 is  
connected to the inner oil reservoir 1 of the hydraulic cylinder 10  
through a check valve 3. An oil channel 31 is installed between the  
inner oil chamber 41 of the piston rod 4 and the sequential valve  
B. A check valve A2 is installed between the oil channel 31 and  
20    the sequential valve B. A oil channel 311 is installed between the  
sequential valve B and the check valve A2 for being connected to  
the inner oil chamber 41 of the piston rod 41. By changing the  
positions of the check valve A2 and the oil channel 31 and 311.  
When in the working conditions of dump load or light load, the  
25    sequential valve B is closed, thus, the hydraulic oil may enter into

the inner oil chamber 41 of the piston rod 4 from the pumping oil chamber 3 through the check valve A2 so that the piston rod 4 will rise rapidly to a still condition (as shown in Fig. 3). In the still load condition, since the check valve A2 closes the oil channel 31, the sequential valve B will open automatically so that the inner oil chamber 41 of the piston rod 4 is communicated to the inner oil reservoir 1. Thus the inner and outer oil pressures of the oil guiding tube 50 in the piston rod 4 are equal. Thus, no strong still load hydraulic pressure does not exist in the oil guiding tube 50 within the inner oil chamber 41 of the piston rod 4. BY this change, the still load hydraulic pressures of the inner oil reservoir 1 and the inner oil chamber 41 of the piston rod 4 can be adjusted equilibrium by the sequential valve B. Therefore, the problem of breakage of the oil guiding tube 50 of the piston rod 4 and the high cost due to confinement in material are solved.

In the oil return circuit, the inner oil reservoir 1 of the hydraulic cylinder 10 is connected to the inner oil chamber 41 of the piston rod 4 through a check valve 4, then it further passes through a release valve C to the outer oil reservoir 2. After unloading so as to be returned to the original position, the release valve C can be adjusted to a releasing condition, so that the oil return circuit is conducted.

In the overloading protecting circuit, the outer oil reservoir 2 of the hydraulic cylinder 10 passes through a safety valve D to be connected to the pumping oil chamber 3. When the pressure of the

hydraulic cylinder 10 is over a rated pressure, the safety valve will be conducted so that the overload protecting circuit will open automatically.

In the aforesaid hydraulic loop system, the maximum effective oil storing amount of the pumping oil chamber 3 is larger than or equal to the maximum effective oil storing amount of the inner oil chamber 41 in the piston rod so that the hydraulic jack may rapidly rise to a loading position in dummy or light load condition.

In the aforesaid operation, when the piston rod 4 moves forwards, since the pressure of the inner oil reservoir 1 of the hydraulic cylinder 10 reduces abruptly, the hydraulic oil will automatically enter into the inner oil reservoir 1 from the outer oil reservoir 2 through an oil channel D12. Moreover, some hydraulic oil may flow to the pumping oil chamber 3 along the oil channel D1 for waiting that the pump 21 process a hydraulic circulate work. Now, the hydraulic oil can not enter into the inner oil chamber 41 of a saturated piston rod 4. Therefore, a pressure sufficient to actuate the sequential valve B is attained, thus, the hydraulic oil will enter into the inner oil reservoir 4 from the oil channel 31 of the pumping oil chamber 3 through the sequential valve B so that the piston rod 4 ejects a heavy object W to be raised slowly in a lower speed to a preset ejecting position or to a whole rising state (as shown in Fig. 4). The pressure of the sequential valve B may be set in the preset value for actuating.



As shown in Fig. 5, the arm 30 of the jack and the top disk 40 are raised quickly to a preset position to contact with a heavy object W from a still load condition along with the displacement of the piston rod 4 so as to rise the object W.

5 According to aforesaid embodiment, the actuating pressure of the sequential valve B can be set in advance according to the practical requirement, and the sequential valve B can be directly arranged within the rear seat 60 of the jack for forming as a hiding type. However, similarly, the sequential valve B' can be located  
10 outsides, as the embodiment shown in Figs. 6 and 7. The sequential valve B' can be located in a position with a 90 degrees shifting, while the check valves A2, A3 and A4 and oil channel 311 connected the sequential valve B' and the common oil channel 31 are remained unchanged. However, in this condition, a  
15 connecting oil channel 11 (as shown in Fig. 8) is required to be installed between the valve ball <sup>B1'</sup>~~B1~~ and the inner oil reservoir 1. Thus, the adjustable nut B2' of the sequential valve B' is locked to the outer wall of the rear seat 60 of the jack. The aforesaid sequential valve B' can be located outsides for adjusting the  
20 pressure setting of the sequential valve B' conveniently. Therefore, the user may set the pressure easily to suit different requirements in actuation. As a consequence, all the modifications of the sequential valve B is within the scope and spirit of the present invention.

25 Although the present invention has been described with

reference to the preferred embodiments, it will be understood that the invention is not limited to the details described thereof. Various substitutions and modifications have been suggested in the foregoing description, and others will occur to those of  
5 ordinary skill in the art. Therefore, all such substitutions and modifications are intended to be embraced within the scope of the invention as defined in the appended claims.